

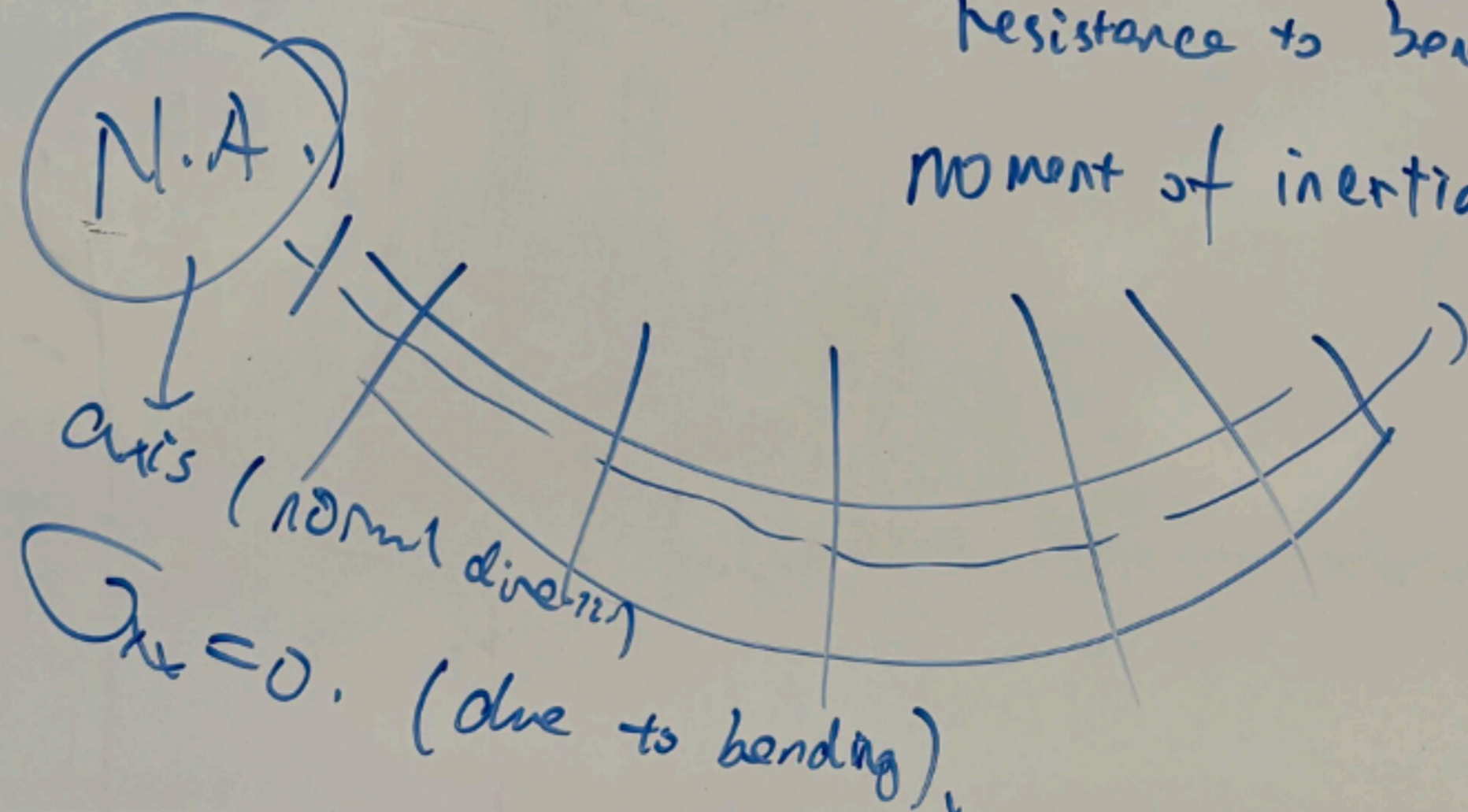
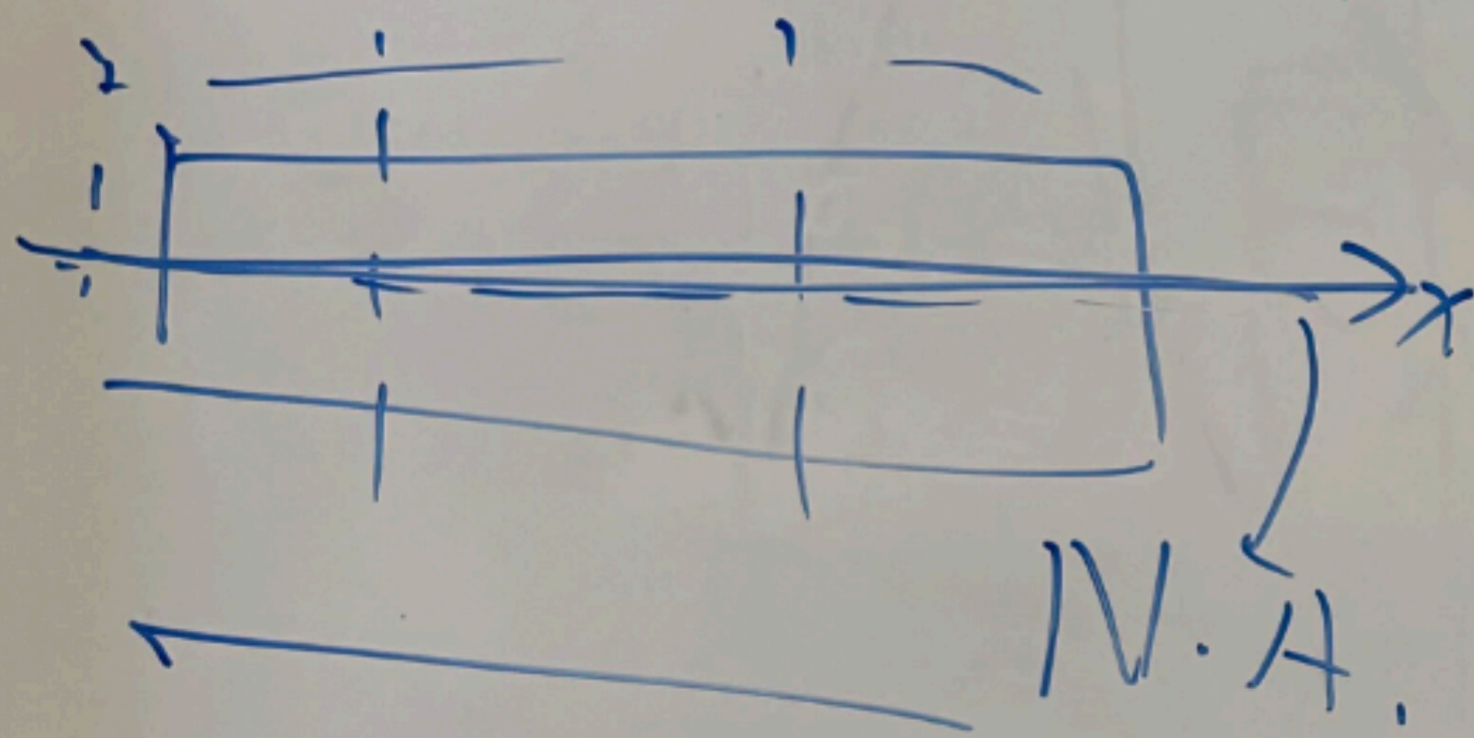
Problem Session #2

- Euler-Bernoulli beam theory ^{MESO}

→ Stress function

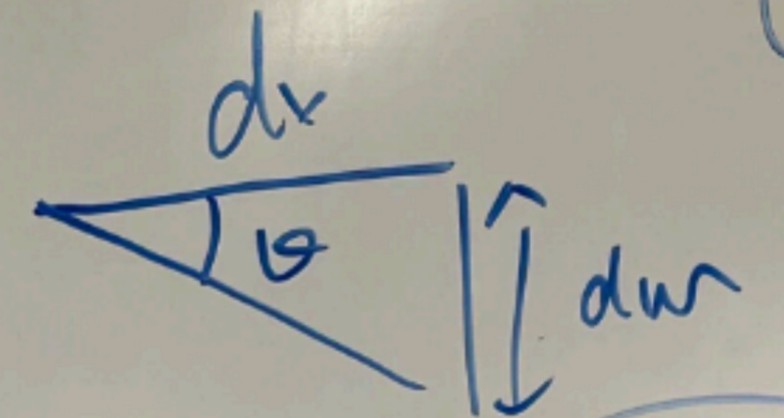
(code for HW #2)

lecture notes Pg. 16 onwards



Assumptions

1. plane surfaces (I to NA.)
2. deflections are small.
3. No normal stress in NA.



$$\tan \theta \sim \theta = \frac{dw}{dx}$$

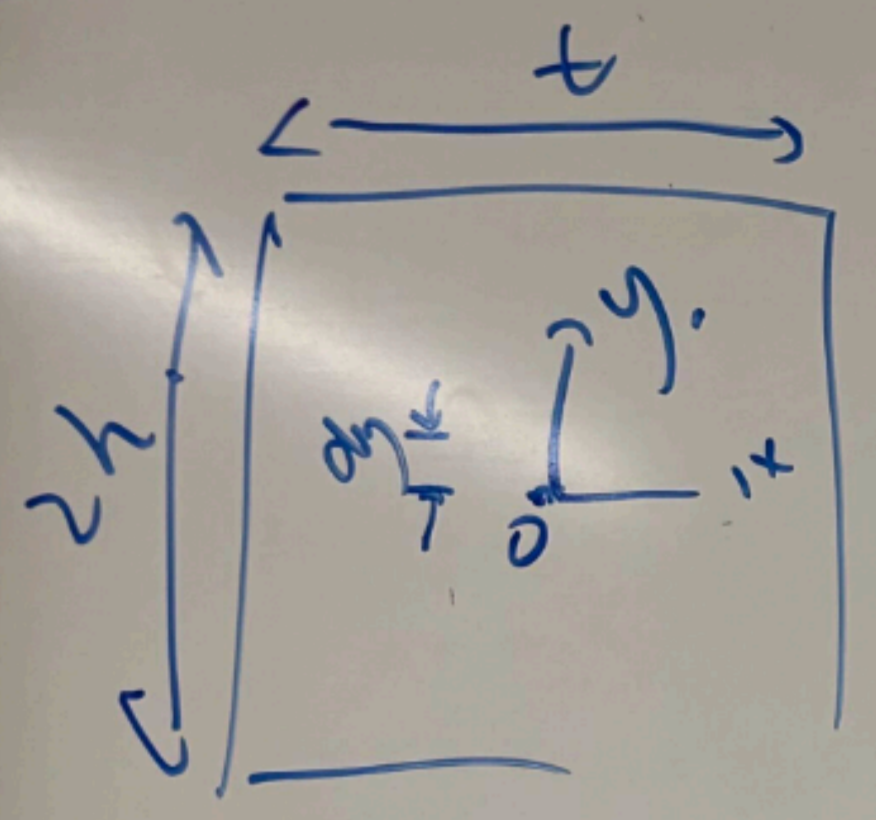
Curvature (k) = $\frac{d\theta}{dx} = \frac{d^2w}{dx^2}$

$$Q = \frac{\text{bending moment}}{\text{resistance to bending}} = \frac{M}{k_s} \rightarrow \text{inertia}$$

moment of inertia: $\int r^2 dm$ $\left(\begin{matrix} k_s \\ EI \end{matrix} \right)$

$$M = EI_z \frac{d^2w}{dx^2}$$

Example #0



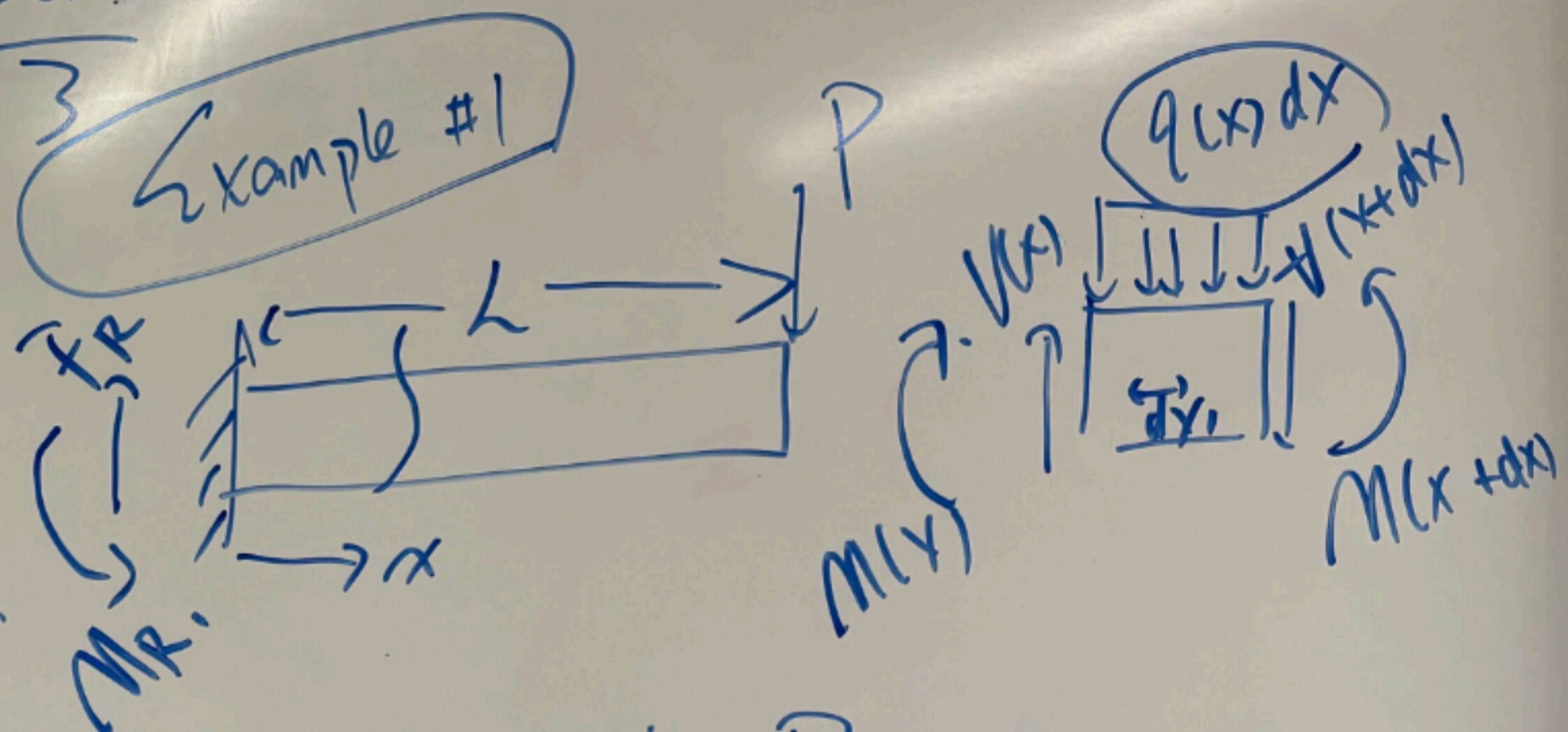
Second moment of area: $I_z = \int y^2 dA$.

$$I_z = \int_{-h}^{+h} y^2 t dy = \frac{2th^3}{3}$$

$$k_s = EI_z = \frac{2Et^3}{3}$$

What do we want?
 $V(x)$. $M(x)$.

$$Q = \frac{M}{EI_z} = \frac{d^2 w}{dx^2} \rightarrow M = EI_z \frac{d^2 w}{dx^2}$$



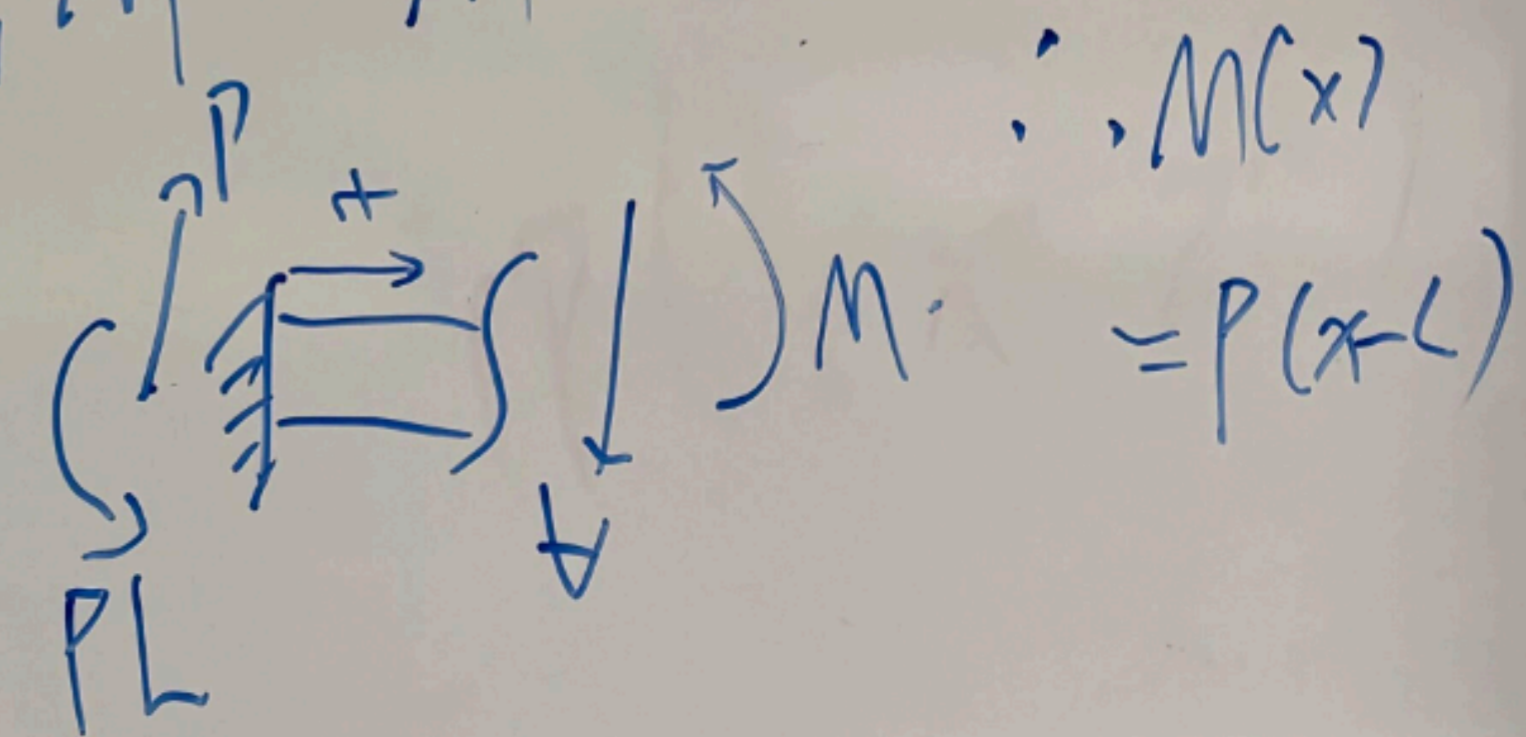
Governing equations:
 { balance of force.
 balance of moment.

$$\frac{dV}{dx} = -q(x)$$

$$\frac{dM}{dx} = V(x)$$

$$\sum F = 0; V = P$$

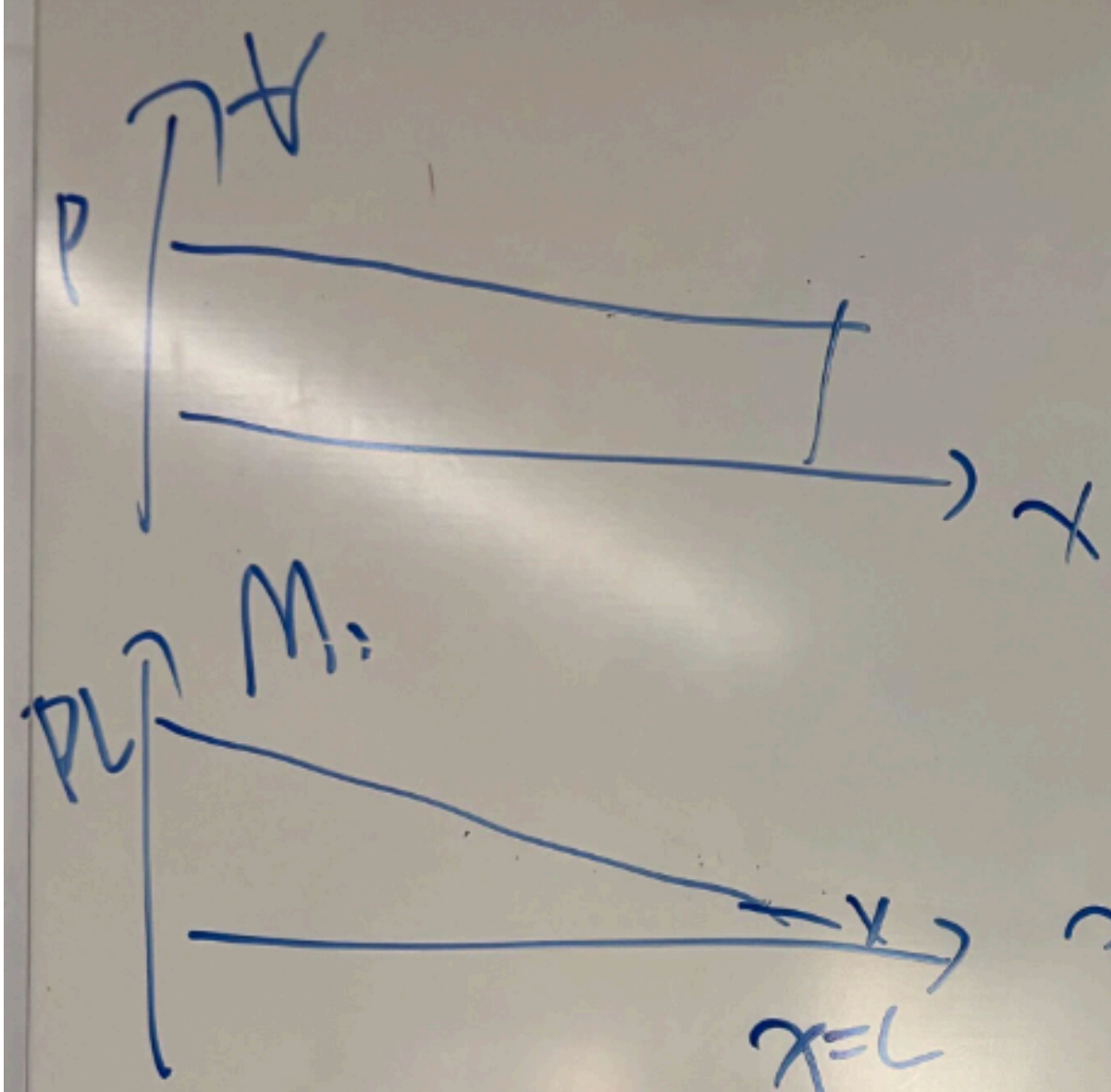
$$\sum M = 0; M + PL = Vx$$



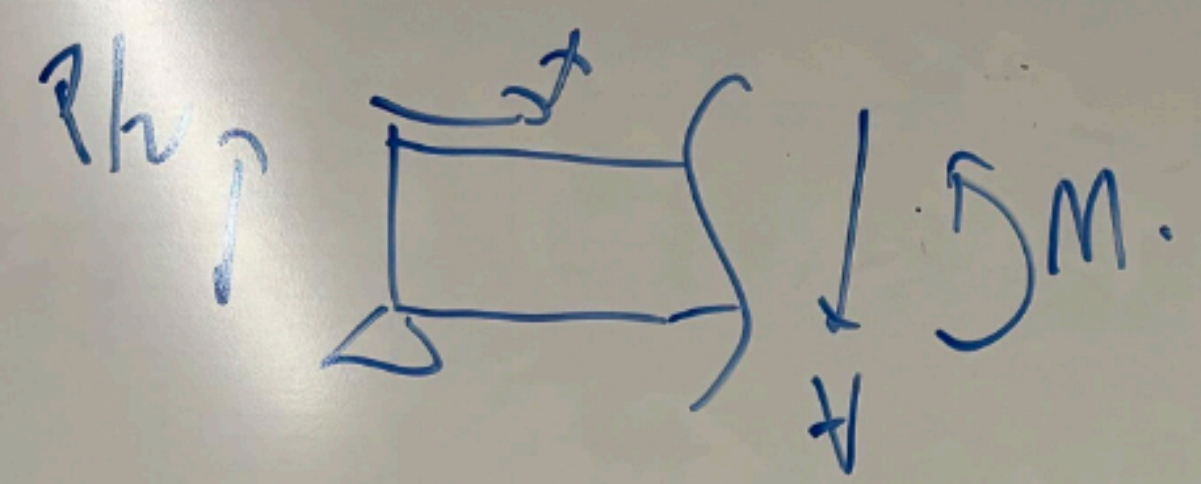
Equilibrium:

$$q(x) = -EI_z \frac{d^4 w}{dx^4}$$

B.C.s? ϕ

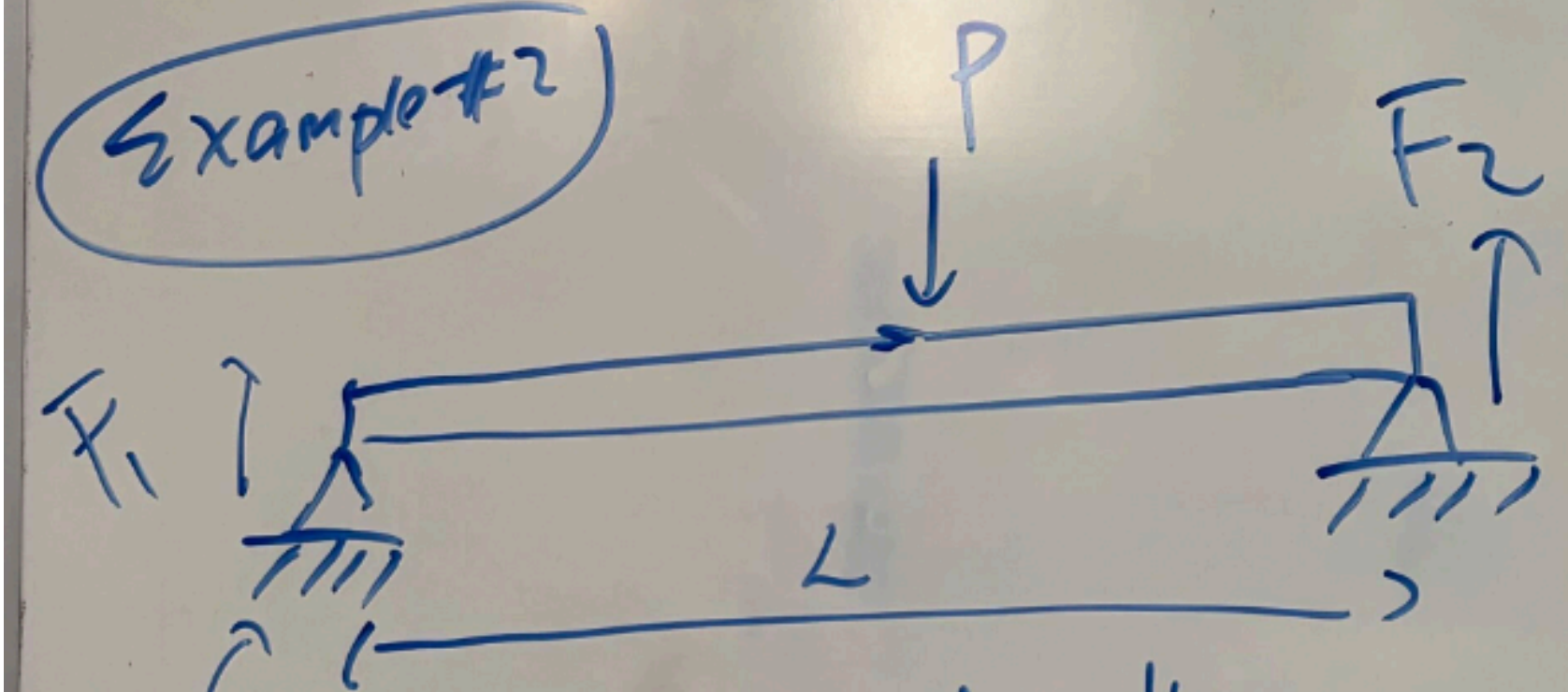


left side $x < L/2$

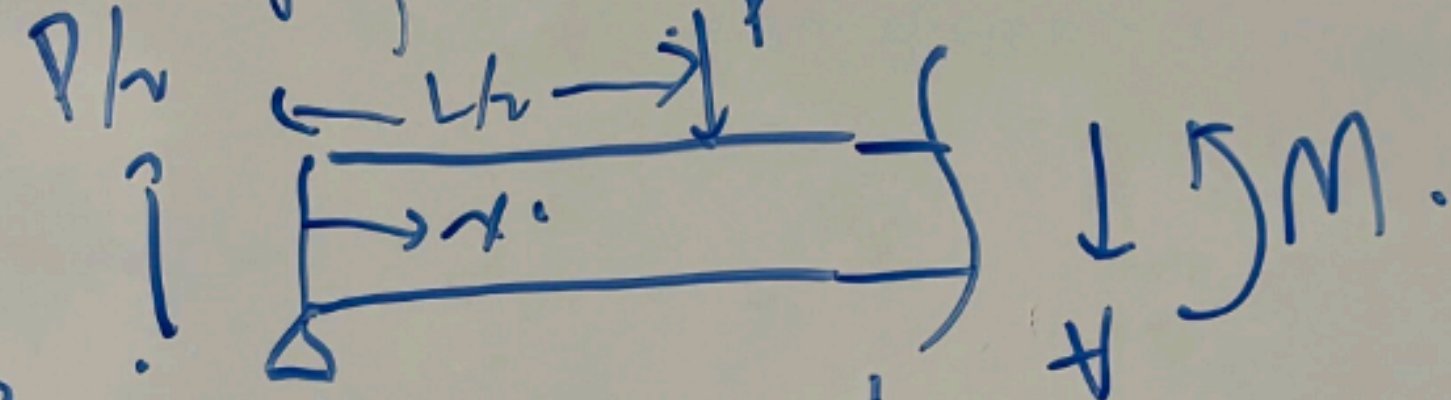


$\sum F = 0: V = P/2$
 $\sum M = 0: M(x) = Vx = Px/2$

Example #2



right side $x > L/2$

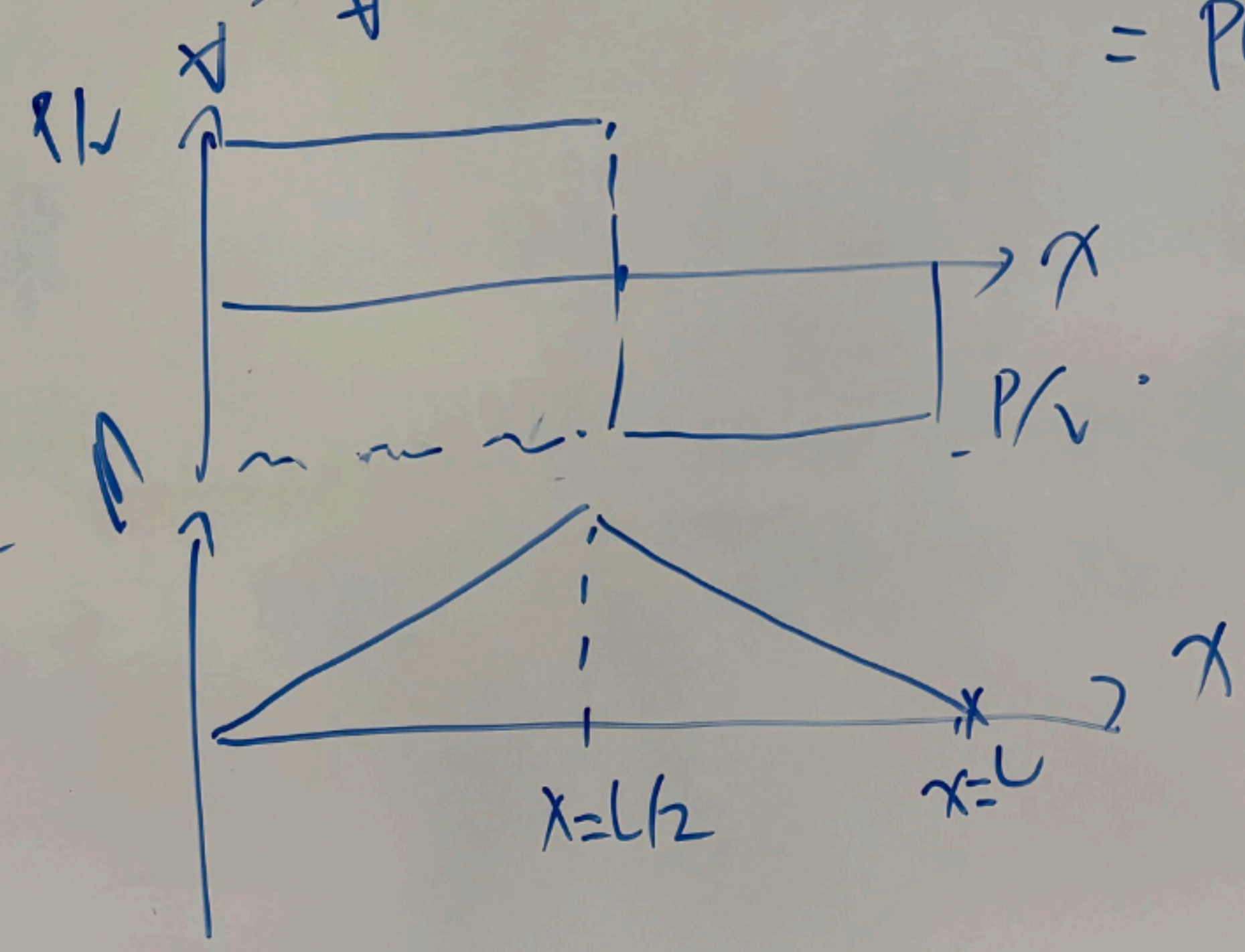


$\sum F = 0: V = -P/2$
 $\sum M = 0: M(x) = \frac{P/2}{2} + V(x - L/2) = \frac{P(L-x)}{2}$

freely supported ends,
 (no moment)

$F_1 = F_2 = P/2$

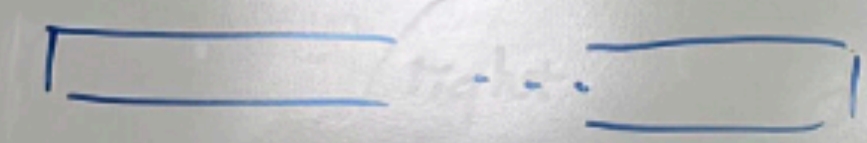
*** ONLY provide reaction force
 DO NOT resist bending



Stress function approach

Lecture note #5 Pg. 11 ~

- Solve problems more accurately "probably".



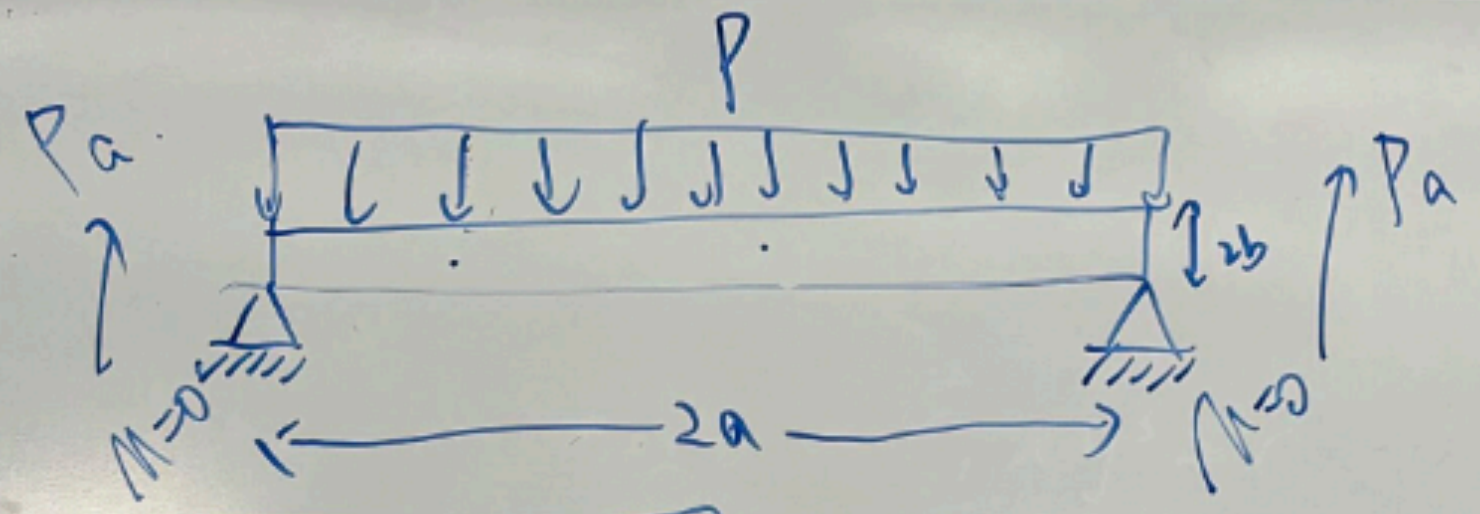
- NOT exact. - Weak B.C.s

HW #2

files / Homework / 5522a.m

Barber

generalized Hooke's law, $\sigma_{ij} = C_{ijkl} \epsilon_{kl}$



$$\begin{aligned} q(x) &\propto x^{0 \dots n} \\ \psi(x) &\propto x^2 \\ M(x) &\propto x^2 \end{aligned}$$

Governing eqn.

$$\nabla^4 \phi = 0$$

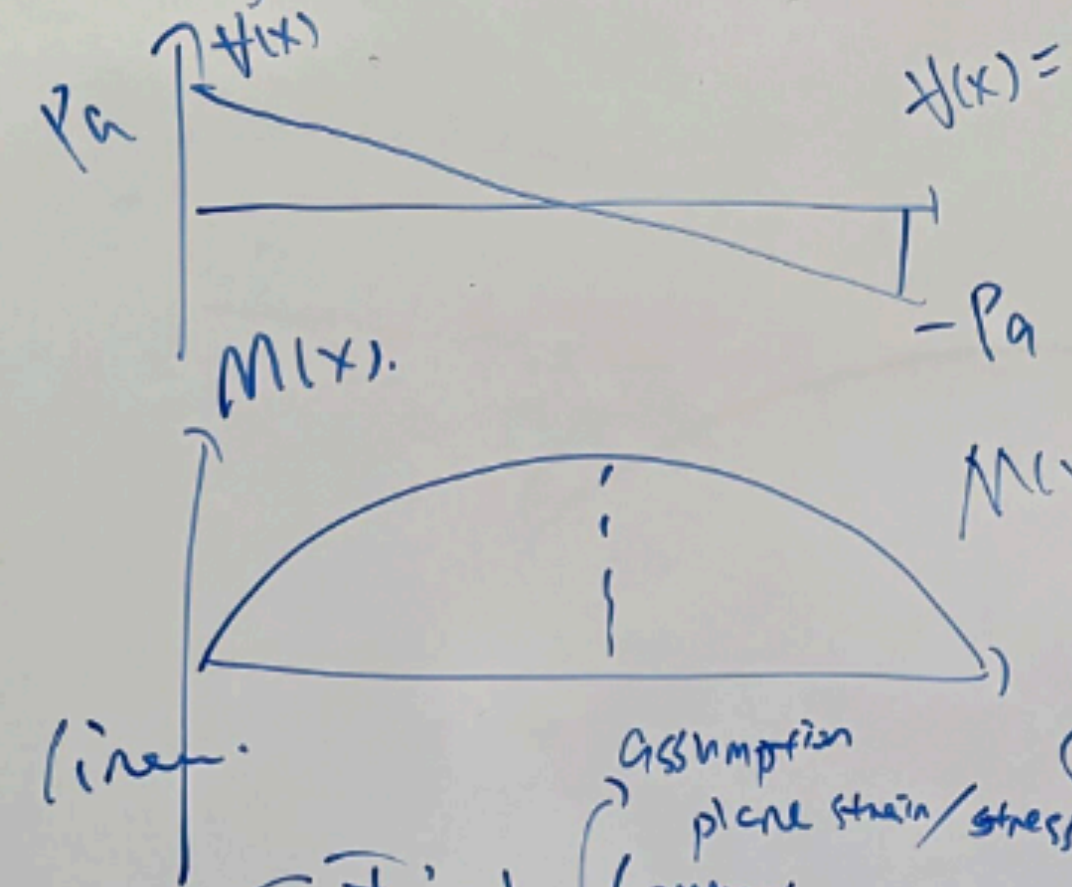
$$= C_1 x^2 + C_2 x + \dots + C_8 y^5$$

$$\sigma_{xx} \propto M(x) \cdot y \rightarrow \phi \propto x^{n+5}$$



$$y \leftarrow \epsilon \text{ (deformation)}$$

Using Euler-Bernoulli beam theory.



$$\psi(x) = -Px^2$$

$$M(x) = \frac{1}{2} P(a^2 - x^2)$$

$$\sigma_{xx}(x,y) = -\frac{M(x)y}{I_z} = \frac{3P}{4b^3} (x^2 - a^2)y$$

Final

Assumption: plane strain/stress

Comment:

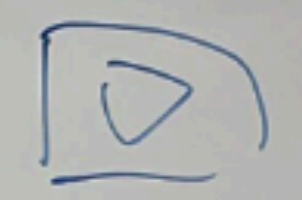
$$\begin{aligned} \sigma_{xx}(x,y) &= \dots \rightarrow \epsilon_{xx}(x,y) \dots \rightarrow \text{disp. } u_x(x,y) \\ \sigma_{yy}(x,y) &= \dots \rightarrow \epsilon_{yy}(x,y) \dots \rightarrow \text{disp. } u_y(x,y) \\ \sigma_{xy}(x,y) &= \dots \rightarrow \epsilon_{xy}(x,y) \dots \end{aligned}$$

$$\phi = C_1 x^2 + C_2 xy + C_3 y^2 + \dots + C_{18} y^5$$

$S_{xx} : \sigma_{xx} = \phi_{,yy}$
 $S_{yy} : \sigma_{yy} = \phi_{,xx}$
 $S_{xy} : \sigma_{xy} = -\phi_{,xy}$

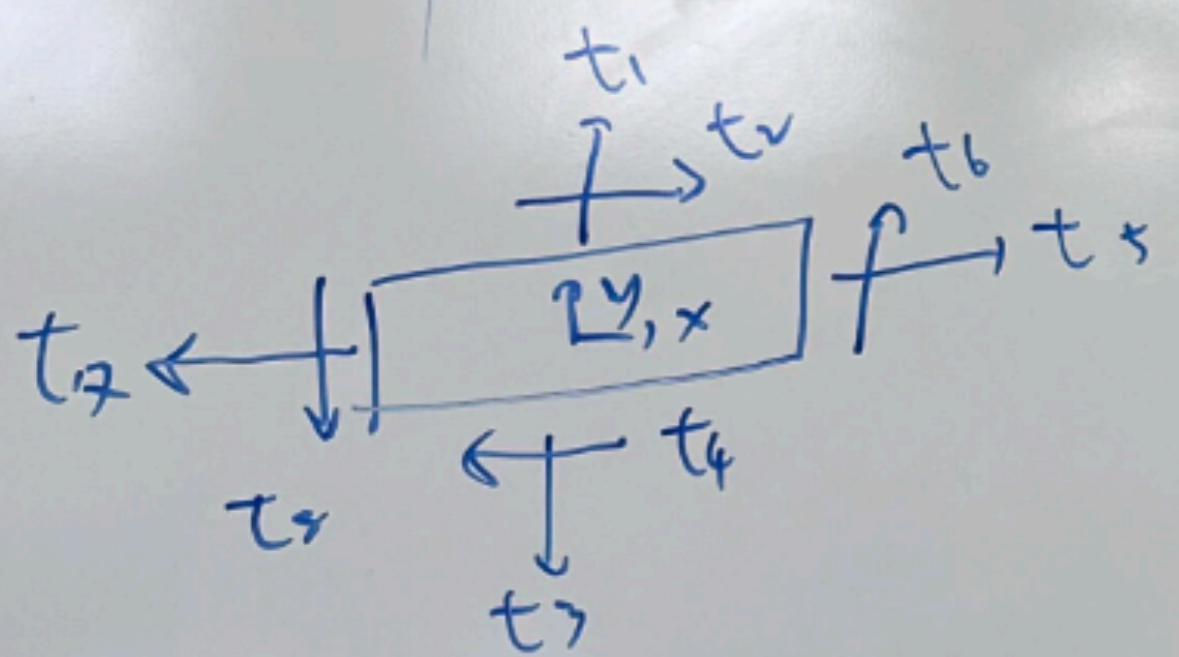
$$T_j = \sigma_{ij} n_i$$

$t_1 = \dots x^3 + \dots x^4 \dots$
 $t_2 = \dots x^4 \dots$
 $t_3 = \dots x^3 \dots$
 $t_4 = \dots x^4 \dots$

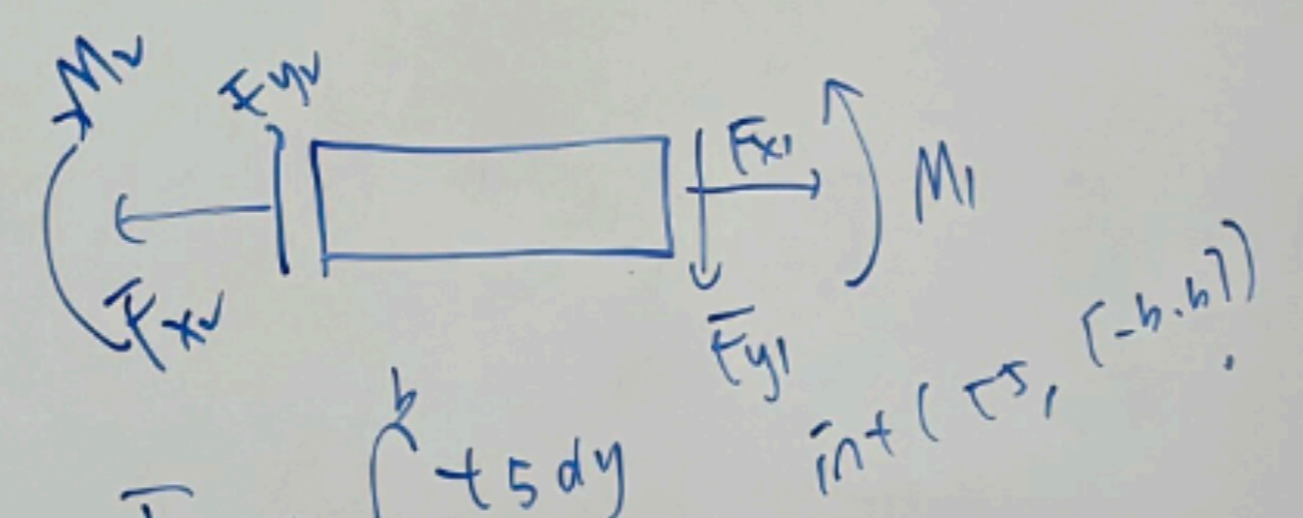


$S_1 = 0 \quad S_2 = 0 \quad S_3 = 0 \quad S_{4+p} = 0$
 $S_5 = 0 \quad S_6 = 0 \quad S_7 = 0 \quad S_8 = 0$
 $S_9 = 0 \quad S_{10} = 0 \quad S_{11} = 0 \quad S_{12} = 0$
 $S_{13} = 0 \quad S_{14} = 0 \quad S_{15} = 0 \quad S_{16} = 0$
 $b_1 = 0 \quad b_2 = 0 \quad b_3 = 0$
 $*_1 \quad F_{x1} = 0 \quad M1 = 0 \quad F_{y1} - pa = 0$
 $F_{x2} = 0 \quad M2 = 0 \quad F_{y2} + pa = 0$

$\phi(x,y) = -\frac{P}{4b^3} (12x^2 b^3 + \dots)$
 $\sigma_{xx}(x,y) = -\frac{P y}{2b^3} \dots$
 $\sigma_{yy}(x,y) = \dots$
 $\sigma_{xy}(x,y) = \dots$



- $t_1 : \sigma_{yy}(x, y=b)$
- $t_2 : \sigma_{xy}(x, y=b)$
- $t_3 : \sigma_{yx}(x, y=-b)$
- $t_4 : \sigma_{xy}(x, y=-b)$
- $t_5 : \sigma_{xx}(x=a, y)$
- $t_6 : \sigma_{xy}(x=a, y)$
- $t_7 : \sigma_{xx}(x=-a, y)$
- $t_8 : \sigma_{xy}(x=-a, y)$



$F_{x1} : \int_{-b}^b t_5 dy$
 $F_{y1} : \int_{-b}^b t_6 dy$
 $M1 : \int_{-b}^b t_5 y dy$
 $F_{x2} : \int_{-b}^b t_7 dy$
 $F_{y2} : \int_{-b}^b t_8 dy$
 $M2 : \int_{-b}^b t_7 y dy$

$$v_{ij} = \frac{\text{force } j}{\text{area } i}$$